

COPECOM - Control potential of different operating methods in small-scale wood pellet combustion

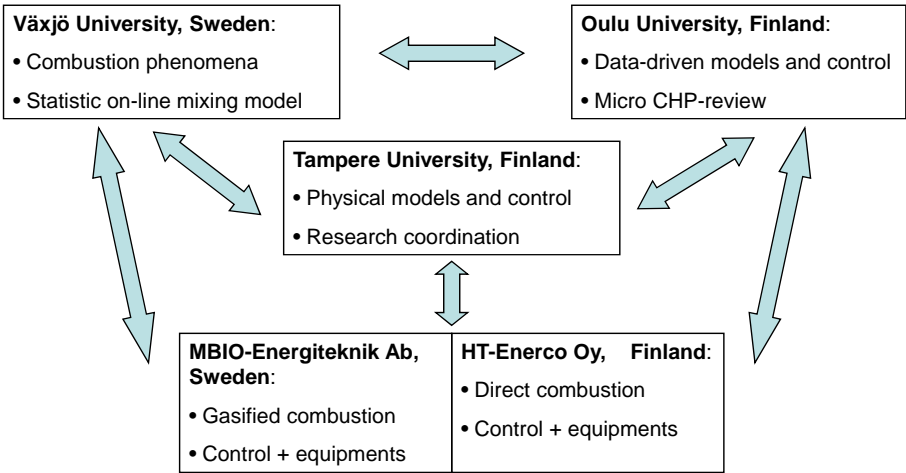
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SSC Conference
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COPECOM Research focus and targets

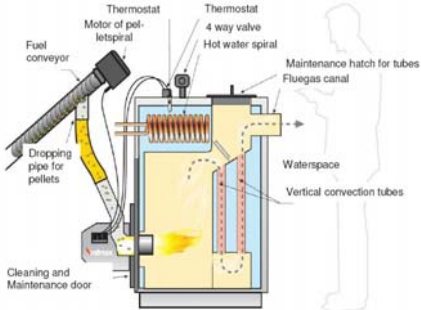
- Control potential of pellet burners – research on gasified and direct combustion, *experimental*
- Physical + data-based modelling and control development, *experimental/theoretical*
- An analysis of integrated small-scale power production and storage, *theoretical*

Research consortium

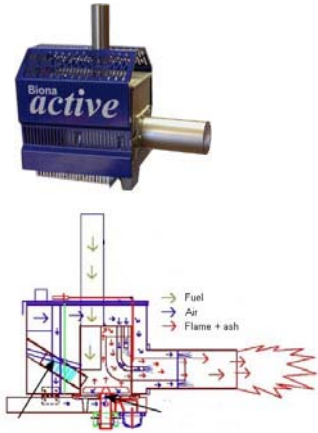


Burners

Velmax



Biona



The Finnish subproject



Reference tests

6 hours, according to EN 303-5

| | Biona | Velmax |
|--------------------------------------------------------------------|----------------|---------------|
| Boiler output, kW, (mean value \pm std) | 15.3 \pm 2.7 | 15.3 \pm 4 |
| Burner output (fuel), kW, (mean value) | 19.7 | 17.5 |
| Total efficiency, %, (from fuel to water) | 78 | 87.4 |
| O ₂ , %, (mean value \pm std) | 5.4 \pm 0.7 | 8.2 \pm 1.3 |
| CO, ppm (mean value @ 10% O ₂ \pm std) | 184 \pm 92 | 191 \pm 588 |
| NO _x , ppm, (mean value @ 10% O ₂ \pm std) | 73 \pm 4.2 | 97 \pm 11.5 |

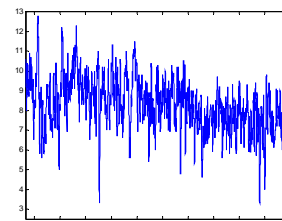
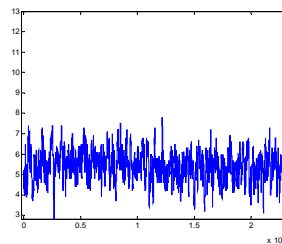
Reference tests

6 hours, according to EN 303-5

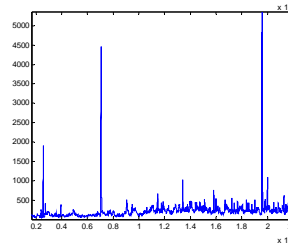
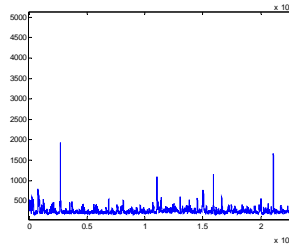
Biona

Velmax

O₂ [vol.-%]



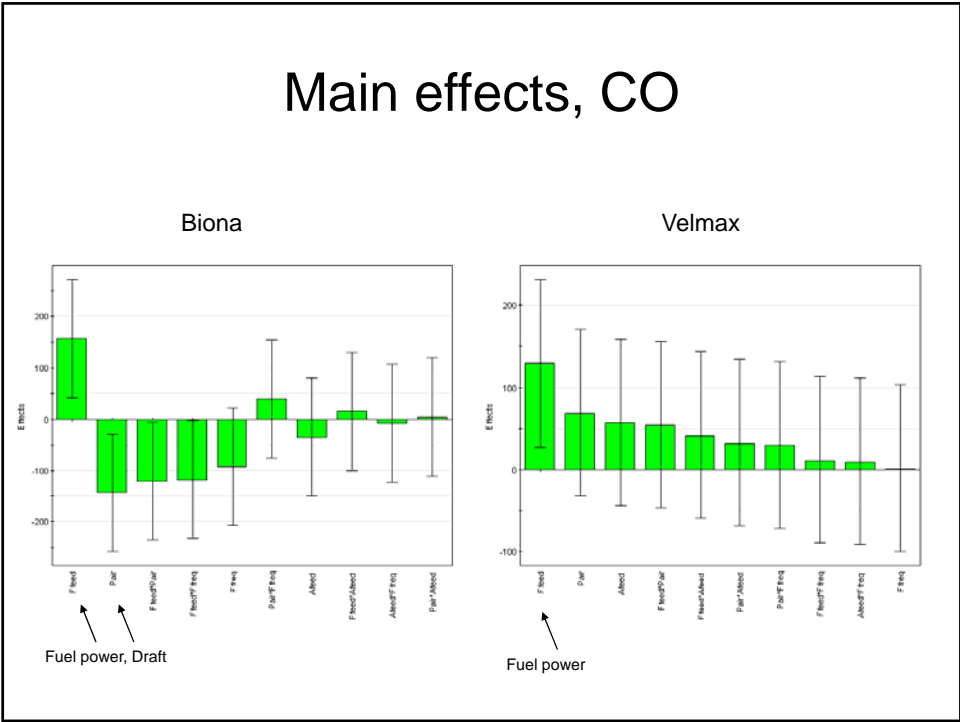
CO [ppm]



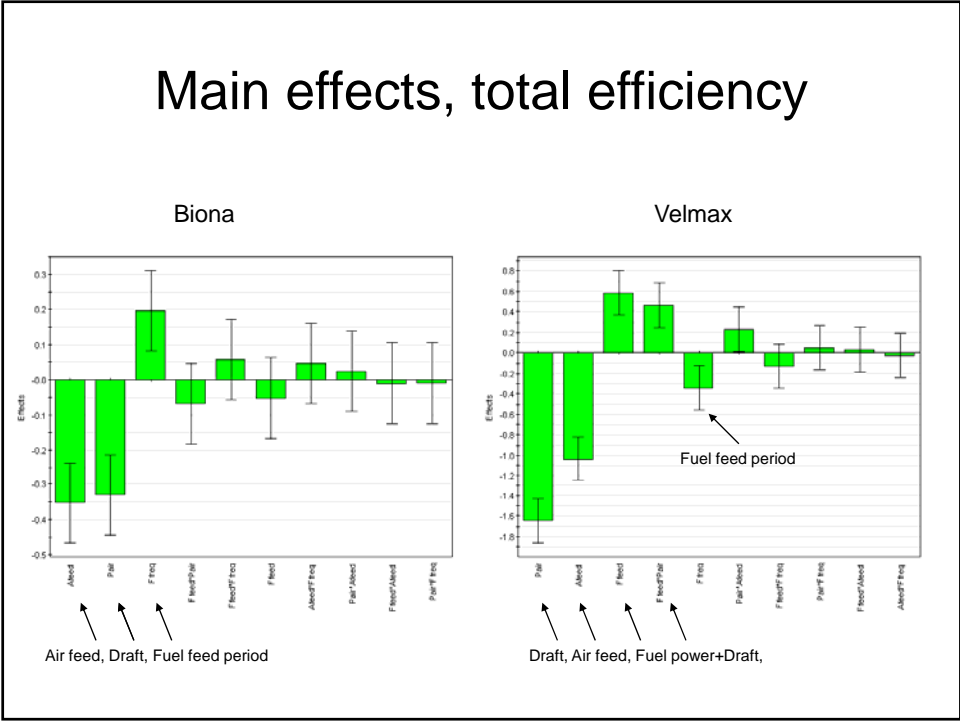
Design of Experiments

- In search for effects and interactions
- Selected control variables
 - fuel power
 - draft
 - air feed
 - fuel feeding period
- Full factorial design
- Partial Least Square (PLS) modelling

Main effects, CO



Main effects, total efficiency



Optimised combustion

Biona, results with tuned control variables

| | Before | After tuning |
|--------------------------------------------------------------------|----------------|----------------|
| Boiler output, kW, (mean value \pm std) | 15.3 \pm 2.7 | 16.5 \pm 0.4 |
| Burner output (fuel), kW, (mean value) | 19.7 | 19.7 |
| Total efficiency, %, (from fuel to water) | 78 | 84 |
| O ₂ , %, (mean value \pm std) | 5.4 \pm 0.7 | 3.8 \pm 0.6 |
| CO, ppm (mean value @ 10% O ₂ \pm std) | 184 \pm 92 | 420 \pm 124 |
| NO _x , ppm, (mean value @ 10% O ₂ \pm std) | 73 \pm 4.2 | - |
| Fuel feed ratio | 47% | 47% |
| Pressure, Pa | 15 | 10 |
| Air feed, m ³ /h | -50 | 43 |
| Fuel feed period, s | 6 | 12 |

Optimised combustion

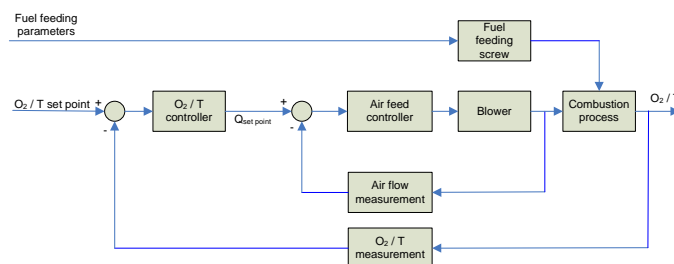
Velmax, with optimal values of control variables

| | Before | After optimisation |
|--------------------------------------------------------------------|---------------|--------------------|
| Boiler output, kW, (mean value \pm std) | 15.3 \pm 4 | - |
| Burner output (fuel), kW, (mean value) | 17.5 | -13 ! |
| Total efficiency, %, (from fuel to water) | 87.4 | - |
| O ₂ , %, (mean value \pm std) | 8.2 \pm 1.3 | 7.3 \pm 0.9 |
| CO, ppm (mean value @ 10% O ₂ \pm std) | 191 \pm 588 | 40 \pm 14 |
| NO _x , ppm, (mean value @ 10% O ₂ \pm std) | 97 \pm 11.5 | 94 \pm 1 |
| Fuel feed ratio | 29% | 20% |
| Pressure, Pa | 15 | 10 |
| Air feed, m ³ /h | 30 | 25 |
| Fuel feed period, s | 12 | 3.5 |

Discussion of experiment results

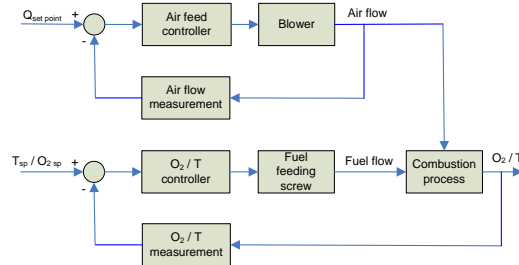
- Explored control variables have effect on combustion
- Stable and proper chosen values for variables important
- Air feed control disturbs combustion (primary + secondary air changes simultaneously)
- Combination of optimal values can reduce emissions and increase total efficiency

A control concept



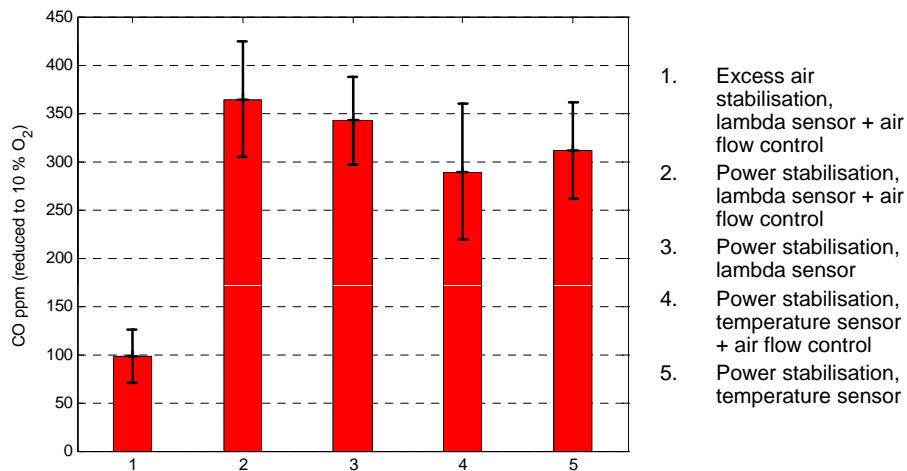
- Excess air stabilisation
- Results in power fluctuation
- Not suitable as the primary and secondary air feeds cannot be separated with burners with one air fan
- Note: Constant air flow do not stabilize the combustion

Proposed control concept



- Power stabilisation
- Settles the fuel feed to air feed
- Enables power level control
- Both Velmax and Biona (with constant power)

CO with different control strategies (Velmax)



Conclusions

- Requirements for clean combustion
 - Draft
 - Air feed
 - Fuel feed
 - Other parameters (e.g. fuel feeding period)
- The combustion properties vary with different systems
- Active control can compensate variations in combustion conditions
- Controllable process and proper instrumentation is needed
- Emission and efficiency requirements determines the automation level



Modelling approach



Fundamental thoughts behind approach:

- Domestic pellet burners may not be of optimal design – nor operating under optimal conditions – for environmentally friendly combustion, though they are carbon-dioxide neutral
- The emissions of unburned hydrocarbons may be radically reduced by the application of modern control technology
- To achieve this, a model that can avoid sudden peak values is necessary.



Two (main) questions:

- 1: What are the average values?
- 2: How big is the variation?

Hypothesis:

The variation in analysis can probably be related to imperfections in the mixing along the gas pathway.

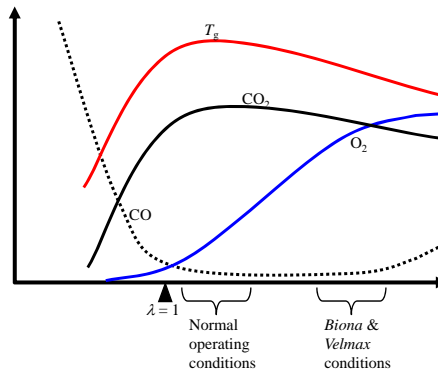


Figure 10: Over all behavior of oxygen, unburnt and temperature in combustion applications as functions of the air factor



Results:

- 1: Mean values measured: TOO HIGH!!
- 2: Variation in CO (meas):
 Biona 455, $\sigma = 219$ ppm
 Velmax 404, $\sigma = 279$ ppm
 (Biona 455, $\sigma = 220$ ppm)
 (Velmax 400, $\sigma = 280$ ppm)

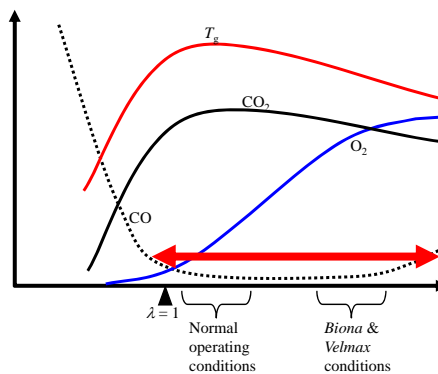


Figure 10: Over all behavior of oxygen, unburnt and temperature in combustion applications as functions of the air factor



Results:

- 1: Mean values: TOO HIGH!!
- 2: Variation in CO (meas):
 Biona 455, $\sigma = 220$ ppm
 Velmax 400, $\sigma = 280$ ppm

Model results:

- Biona 600, $\sigma = 400$ ppm
- Velmax 0, $\sigma = 300$ ppm

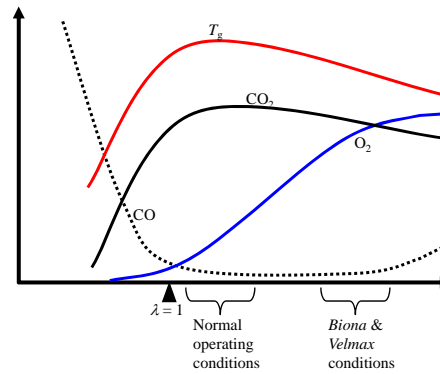


Figure 10: Over all behaviour of oxygen, unburnt and temperature in combustion applications as functions of the air factor

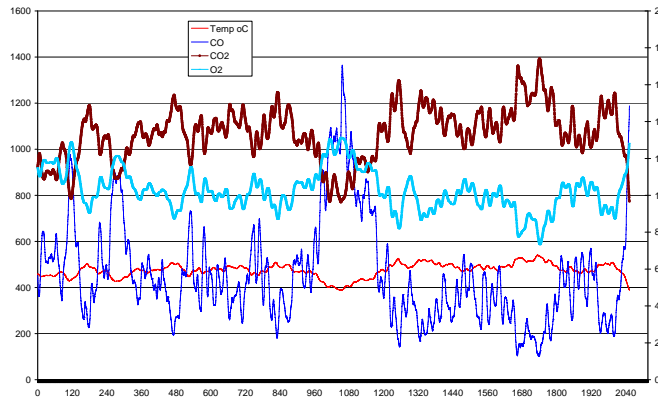


The model results seem to indicate that statistical modelling can be used to reproduce the variations in CO as well as its over-all behaviour.

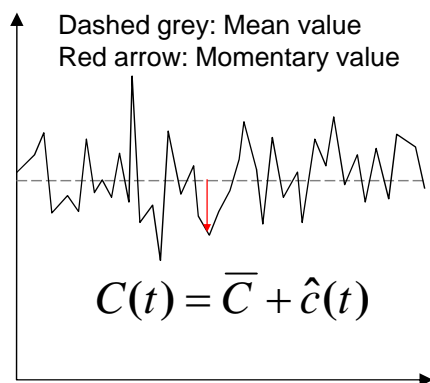
- How does the statistical model work?



Time data for gas analysis, 35 min exp:



Time data for local gas analysis:



The mean value can be calculated from a simple mass balance.

The momentary value needs more consideration...



Resolved data for gas analysis:

Two major ways to attack the problem:

- 1) Make time averages and solve the Navier-Stokes and continuity equations for node points in space
This yields space-resolved concentration distributions and is the classical method used in CFD modelling.
- 2) Assume constant averages across macroscopic volumes and introduce functions for the successive development of the distribution as a function of time.
This yields approximate distributions at the exit from each computational volume.



Resolved data for gas analysis:

The third way to attack the problem:

- 3) Solve the Navier-Stokes and continuity equations for node points in space and following turbulent disturbances through time.
This yields space-and-time-resolved concentration distributions and is quite CPU-intensive and time consuming.

This is probably not reasonable for on-line control...



Resolved data for gas analysis:

The second way is sufficient in this case, since the aim is to reproduce the general behaviour of the gas analysis, i.e. the frequency (and thus the total, time-integrated amounts) of harmful emissions.



Resolved data for gas analysis:

The main weaknesses of this treatment are that it must be based on correlations rather than on thorough models and that it can frequently be applied only to the gas phase while the solid phase must be extremely simplified.

Step 1

Define a time series for the fuel-air mixture at the inlet



Resolved data for gas analysis:

The secondary is sufficient in this case as the air is based on product the general behavior of the gas analysis that it can frequently (and thus the total time integrated amounts) of harmful emissions.

Step 1

Define a time series for the fuel-air mixture at the inlet

Step 2

Modify the time series according to predetermined functions



Resolved data for gas analysis:

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Step 1

Define a time series for the fuel-air mixture at the inlet

Step 2

Modify the time series according to predetermined functions

Step 3

Modify the time series with respect to additional air



Resolved data for gas analysis:

The main weaknesses of this treatment are that it must be based on correlations rather than on thorough models and that it can treat only the gas phase while the solid phase must be extremely simplified.

Step 1

Define a time series for the fuel-air mixture at the inlet

Step 2

Modify the time series according to predetermined functions

Step 3

Modify the time series with respect to additional air

Step 4

Iterate 2 and 3 over all volumes until finished



Find one parameter to be followed:

The air factor – λ – has the disadvantage of being defined on an open interval, $[0, \infty[$, which makes it difficult to use.

Use the mixing factor instead: $f_{mix,air} = \frac{\lambda \cdot m_{air,stoichiometric\ need}^0}{\lambda \cdot m_{air,stoichiometric\ need}^0 + 1 + m_{inert}}$

where m_{air}^0 is the stoichiometric need of air in kg/kg wet fuel and m_{inert} is the momentary amount of flue gases in kg/kg wet fuel.



Set up an initial time series:

At the inlet, there can be only fuel ($f_{\text{mix}} = 0$) or air ($f_{\text{mix}} = 1$).

Hence, we define a randomized series 1100010011110100100... with the correct mean value and with a frequency representing a typical frequency of small-scale turbulence for the geometry.

The frequency is chosen as $\nu = \frac{\bar{U}}{d_p}$, where \bar{U} is the mean gas velocity and d_p is the mean particle diameter.



Modify the time series successively:

As the series of eddies passes the physical volumes, the small eddies dissipate, representing a shift from higher to lower frequencies.

This is modelled by digital filtering using the response time function – which is the frequency characteristic of the volume – as the filter function.

Filtering does not affect the mean value but only the frequency spectrum.



Insert additional air at inlets:

Additional air – secondary, tertiary... – adds momentum and adds mass.

Momentum is added by decreasing the Peclet number in the filter functions (i.e. increasing the effective, turbulent, diffusion).

Mass is added by randomly inserting 1's into the time series so as to change the mean value of the mixing factor f_{mix} in accordance with the physical reality.

To represent the increase in velocity, the total time is maintained as the number of eddies increase. Thus the frequency increases.



Iterate:

The time series – successively with more and more points as more and more air is injected – is followed downstream.

For each mixing factor value in the time series, the change from inlet to outlet is evaluated and recalculated into gas composition.

These values yield a change in gas heating value, and thus a heat release, in that volume. This is used to calculate a new gas temperature and a residence time in the volume. This is iterated in each volume with respect to the heat losses.

After the final volume, a time series for the exit gas is obtained.



The air factor:

From the last gas volume, a time series representing the mixing factor $f_{mix}(t)$ is now known, as is the gas temperature.

From $\lambda = \frac{f_{bo}}{1 - f_{mix,air}} + \frac{1}{m_{air,st}^0} \cdot \frac{f_{mix,air}}{1 - f_{mix,air}}$, this can be used to calculate $\lambda(t)$.

Knowing $\lambda(t)$, the gas composition can be readily calculated.



Gas composition when $\lambda < 1$ (IV):

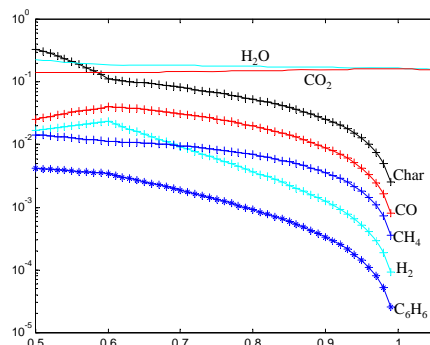


Figure 1: Distribution of hydrocarbons as a function of the air factor at sub-stoichiometric conditions

Sample calculation showing the fractions of the different components as a function of λ at sub-stoichiometric conditions.



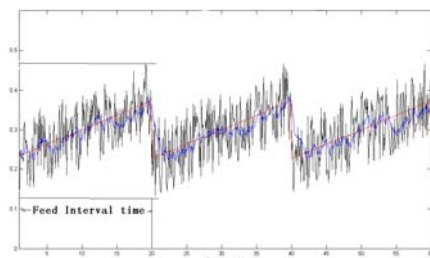
Gas composition when $\lambda \geq 1$:

In case of over-stoichiometric conditions, simple stoichiometry is used to calculate the gas composition from the air factor.



The Biona burner, gasifier volume:

In the Biona, pellets are fed batch-wise creating a saw-tooth pattern for the air factor in the primary zone:



Red: λ mean baseline

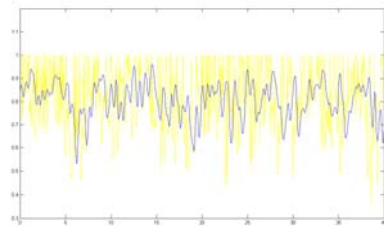
Black: λ randomised to input

Blue: λ at exit of volume 1



The Biona burner, combustion chamber:

The last air in the Biona is injected just at the inlet to the boiler combustion chamber:



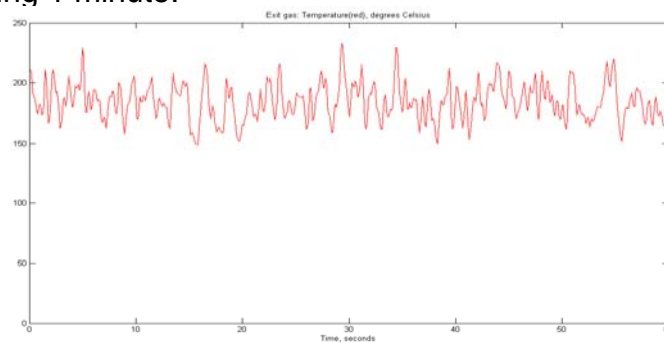
Yellow: f_{mix} at air inlet

Blue: f_{mix} at fireplace exit
This represents the flue gas



The Biona burner, results:

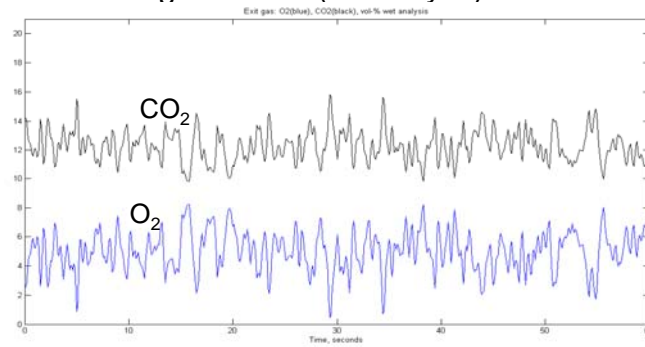
The diagram shows the simulated gas temperatures at the boiler exit during 1 minute:



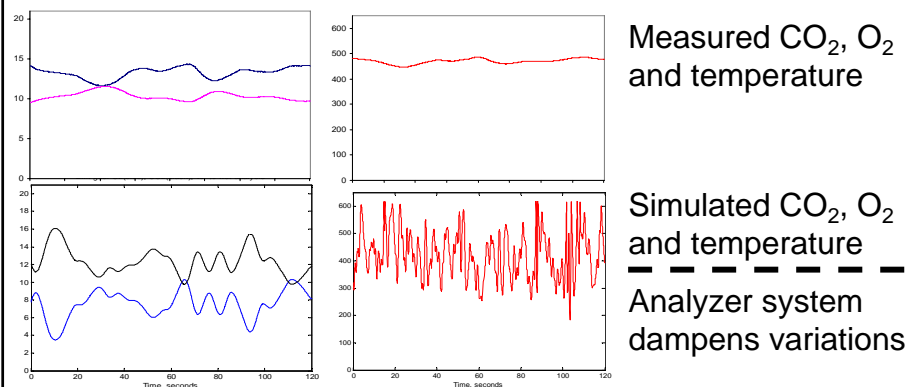


The Biona burner, results:

The diagram shows the simulated gas analysis (CO_2 and O_2) at the boiler exit during 1 minute (*wet analysis*):



The Biona burner, comparison, 2 min:





The Biona burner, five simulations:

| $O_{2,mean}$ | σ_{O_2} | $CO_{2,mean}$ | σ_{CO_2} | CO_{mean} | σ_{CO} |
|--------------|----------------|---------------|-----------------|-------------|---------------|
| 7.8 | 1.0 | 12.0 | 0.9 | 842 | 459 |
| 8.1 | 1.3 | 11.8 | 1.4 | 676 | 635 |
| 7.6 | 1.6 | 12.3 | 1.5 | 562 | 203 |
| 7.7 | 1.3 | 12.2 | 1.2 | 758 | 371 |
| 7.8 | 1.0 | 12.0 | 1.0 | 557 | 206 |
| 7.8 | 1.3 | 12.1 | 1.2 | 679 | 409 |



Burner vs. Combustion chamber:

Experiments in a Baxi boiler showed incomplete combustion from both burners.

Experiments in an axisymmetric, insulated, combustion chamber also demonstrated incomplete burnout – though better.

Simulations confirm the trend.

Thus, the over-all behaviour of the statistical model seems correct within the limits set by the simplifications used.



The statistical model in general:

The suggested statistical model can:

- Reasonably well predict an over-all gas analysis containing several % O₂ in combination with several hundred ppm CO
- Reasonably well reproduce the variation in CO-emissions as a measure of total unburnt
- Describe how changes in geometry from one to another will influence the over-all mixing and burnout, as long as the changes are not too radical



The statistical model in general:

The suggested statistical model can *not*:

- A priori – i.e. without adoption to experimental values – predict the behaviour of a burner/boiler combination
- Predict effects primarily caused by chemical kinetics
- Extrapolate to very large changes



The statistical model in general:

The suggested statistical model can thus:

- Be used as a predictor in a control system, pointing out the correct direction for adjustments to improve the burnout
- Improve its own predictive power by collecting data successively

COPECOM: Developed control framework

- Temperature measurement information is enriched by mathematical modelling
 - Models to determine emissions and efficiency (UO + TUT)
 - They are used with statistic mixing model to gain information in which direction the process should be driven (VXU)
 - The result is combination of models
 - The enriched information is the foundation for control

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